
About buckling of X-type bracing

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Abstract: In X-type bracing structures generally it does not take into account the structural contribution of the compressed member, since it is assumed to have a negligible compressive strength; it is considered that the stretched member takes the total stress. In the following we analyse as the stretched diagonal, joined in the middle to the compressed one, improves the structural behaviour of this both with respect to the buckling in-plane or out-of-plane of the structure. First we recall the link between the Euler buckling load of a rod free and braced depending on the stiffness k of the brace. Then we analyse the in-plane and out-of-plane buckling of the rod and, for the two situations, we value the increase of the buckling load due to the elastic brace. In the end for both cases we show in what condition the stretched rod causes that the compressed one buckles in the second mode.

Keywords: Steel Bracing Frame, Buckling of Steel Structure, X-Type Diagonal Bracing

1. Introduction

The bracing elements in steel structures have the twofold function to take the structural actions due to horizontal loads, such as wind and earthquake, and to ensure the overall stability of the building to which they belong, so you need to have a high stiffness to limit maximum displacements.

To ensure bracing good stiffness and to control the weights lattice girders with X-type wall bracing are used.

For these girders the structural contribution of the compressed rods is ignored, since it is assumed to have a negligible compressive strength, while the structural function is assigned only to the stretched one.

The purpose of this paper is to define how the stretched rod contributes to reduce the compressed rod effective length, connected to it, to increase its Euler buckling load and therefore the possibility of the compressed rod to effectively contribute to the bracing strength.

We recall from the problem of the compressed rod elastically braced raised in [1] for different cases that was later taken, among others, in [2], [3], [4], [5] with reference to civil and industrial buildings and to steel bridges in [6] [7] [8] [9].

Then we consider the contribution of the stretched rod to the stability of the compressed one as a displacement constraint in-plane and out-of-plane of the structure.

In both situations we look for the condition for which the

effective length of the compressed rod can be considered equal to half of the geometric length.

2. Stability of an Elastically Braced Compressed Rod

With reference to the bracing structure shown in [Fig.1] we outline the compressed diagonal braced in the middle by an elastic constraint [Fig.2].

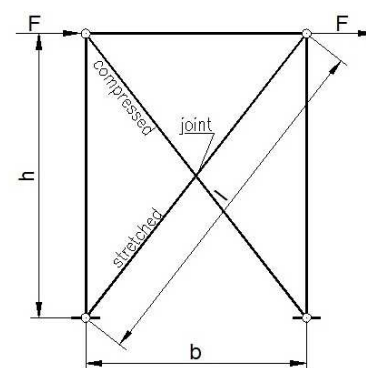


Figure 1. X-type bracing structure

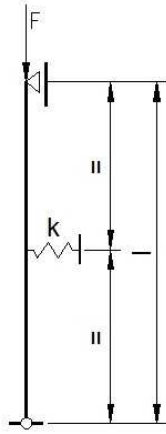


Figure 2. Static scheme for the compressed diagonal

It is well known that, with reference to a changed equilibrium shape and for small displacement $y(z)$, we have:

$$EJy''(z) = -Fy(z) + ky_c z/2 \quad (1)$$

that putting

$$a^2 = F/EJ \quad (2)$$

leads to

$$y''(z) + a^2 y(z) = a^2 ky_c z/2F \quad (3)$$

and, using the boundary conditions $y(0)=0$ e $y(l/2)=y_c$, to the general solution in term of displacement

$$y(z) = (y_c/\sin al/2) (1 - kl/4F) \sin az + ky_c z/2F \quad (4)$$

The further boundary condition $y'(l/2)=0$ leads to

$$(y_c/\sin al/2) (1 - kl/4F) a \cos az + ky_c/2F = 0 \quad (5)$$

and for $y_c k/2F \neq 0$ it becomes

$$al/2 - tg al/2 = 2aF/k \quad (6)$$

and also from [1]

$$(t - tg t)/t^3 = 16P_E/\pi^2 kl \quad (7)$$

in which

$$t = al/2 \quad (8)$$

and P_E is the Euler buckling load of the compressed rod not braced.

Numerically solving the (7) we obtain t and from (8)

$$P_{cr} = P_E 4t^2/\pi^2 \quad (9)$$

where $4t^2/\pi^2$ represents the Euler buckling load increase of the compressed rod due to the elastic constraint.

3. Bracing in-Plane Stability of the Compressed Rod

In the following we want to give an account of what is

generally considered in the project and also admitted by some technical codes [10], [11], for which, with reference to X-type diagonals connected to each other at the middle, the buckling length in the plane of the beam is equal to 0.5l and then

$$P_{cr} = 4 P_E.$$

From the (9) we obtain

$$4P_E = P_E 4t^2/\pi^2 \quad (10)$$

from which results $t = \pi$ and

$$(t - tg t)/t^3 = 0.1015 \quad (11)$$

Now with reference to the (7) and considering $k = 2EA_s/l_s$, where A_s and l_s are the area and the length of the tensile rod respectively, it becomes

$$(t - tg t)/t^3 = 16P_E/\pi^2 (2EA_s/l_s) l_c \quad (12)$$

where l_c is the length of the unbraced compressed rod.

Now from the expression of the Euler buckling load and of the inertia radius

$$P_E = \pi^2 EJ/l_c^2 \quad i^2 = J/A_c$$

the (12) is reduced to

$$(t - tg t)/t^3 = 8\Delta A/\lambda^2 \Delta l \quad (13)$$

with $\Delta A = A_c/A_s$, $\Delta l = l_c/l_s$ and λ is the slenderness of the unbraced compressed rod.

If we equalize the left-hands of (11) e (13) we obtain

$$\Delta A = 0.0127\lambda^2 \Delta l \quad (14)$$

that connects the areas of the stretched and compressed rods with the respective lengths to ensure that the stretched rod carries out a brace such that the effective length of the compressed rod results 0.5l

If we consider ΔA in the interval $60 \leq \lambda \leq 250$, limits between the squat and slender truss, and we remark that for X-type diagonals $\Delta l = 1$ from (14) it derives

$$\Delta A(60) = 45.72 \text{ e } A_s = 2.20\% A_c$$

$$\Delta A(250) = 793.75 \text{ e } A_s = 0.126\% A_c$$

Therefore it is enough a weak constraint to reduce the rod effective length to 0.5l.

For an example we think of a section bar L90x9, 300 cm in length, with area $A_c = 15.5 \text{ cm}^2$ and inertia radius $i_{min} = 1.76 \text{ cm}$; the slenderness is $\lambda = 170$ and so from (14) results

$$A_s = 0.271\% A_c \text{ and } A_s = 0.042 \text{ cm}^2.$$

A rod of 2.3 mm in diameter is enough to efficaciously brace the compressed rod and to impose that its effective length is equal to 0.5l.

4. Bracing Out-of-Plane Stability of the Compressed Rod

In general in the structural design of X-type bracing we ignore the structural behaviour of the compressed diagonal, considered to have negligible capacity, and the whole resistance is entrusted to the stretched diagonal.

However the stretched rod elastically braces the compressed one with a own stiffness k , that depends to its traction force N_s and to its bending stiffness EJ/I^3 .

Also in this case it is possible to refer to the (7)

$$(t-tg\ t)/t^3 = 16P_E/\pi^2 k l$$

where k , from [3], is

$$k = 4N_s/I_s \{1/[1 - \tanh(a_s l_s/2)/(a_s l_s/2)]\} = 4 N_s/I_s \beta \quad (15)$$

where

$$a_s^2 = N_s/EJ_s \quad (16)$$

and

$$\beta = \{1/[1 - \tanh(a_s l_s/2)/(a_s l_s/2)]\} \quad (17)$$

We remark that for $J_s \rightarrow 0$ $\beta = 1$ so, from (17), we obtain the stiffness k of a rope tight from N_s .

Replacing now the (15) in the (7) we have

$$(t-tg\ t)/t^3 = (4/\pi^2 \beta)(\Delta F/\Delta l) \quad (18)$$

in which

$$\Delta F = P_E/N_s \quad (19)$$

and if $\Delta l = 1$ then

$$(t-tg\ t)/t^3 = (4/\pi^2 \beta)\Delta F \quad (20)$$

The determining of t and P_{cr} , from the (9), of the compressed rod elastically braced depends on ΔF e β , that depends on ΔF too, in fact from the (8)

$$t^2 = a_s^2 l^2/4 = (N_s/EJ)l^2/4 \quad (21)$$

and also

$$a_s^2 l^2/4 = (N_s/P_E)\pi^2/4 \quad (22)$$

from which

$$a_s l/2 = (\Delta F)^{-1/2} \pi/2 \quad (23)$$

Then the (17) changes into

$$\beta = \{1/[1 - \tanh(\Delta F)^{-1/2} \pi/2 / (\Delta F)^{-1/2} \pi/2]\} \quad (24)$$

in this way, defined ΔF , we can fix β , t and, from the (9), the new Euler buckling load P_{cr} of the elastically braced compressed rod

$$P_{cr} = P_E 4t^2/\pi^2$$

and the relative new effective length

$$l_0 = (\pi/2t)l_c \quad (25)$$

From the (25) derives that the compressed rod buckle in the second mode, $l_0 = 0.5l_c$, when $t = \pi$ and then from the (20) follows

$$(4/\pi^2 \beta)\Delta F = 0.1013$$

that numerically solved gives rise to

$$\Delta F = P_E/N_s = 0.4 \quad (26)$$

and also to

$$N_s = 2.35 P_E \quad (27)$$

so for $\Delta F > 0.425$ the compressed rod buckles in the second mode.

We remark that also for smaller tight force, $\Delta F = 1$ o $\Delta F = 0.5$, the effective length of the compressed rod is lower than $0.6l_c$.

Now since the (27) links the stress N_s of the tensile rod with the Euler buckling load P_E of the not braced compressed, it is necessary to check if the value of N_s , requisite to secure the brace for the compressed rod, isn't a strength limit for the same stretched rod.

So with reference to the steel design stress f_d and the Euler critical stress $\sigma_{cr,E}(\lambda) = P_E/A_c = \pi^2 E/\lambda^2$ of the compressed rod the (27) changes in

$$f_d = 2.35 \sigma_{cr,E}(\lambda) A_c/A_s = 2.35 \sigma_{cr,E}(\lambda) \Delta A \quad (28)$$

or

$$\sigma_{cr,E}(\lambda) = f_d/2.35 \Delta A \quad (29)$$

Now solving for λ , the slenderness of the not braced compressive rod, we can obtain the value of the length l besides the stretched rod leads efficaciously the function of constraint so that the compressed one buckles in the second mode.

For lower values of λ the stretched rod can't make the brace function as for the value of N_s would exceed the steel mechanical strength.

5. Conclusions

In the X-type diagonal bracing the stretched rod carry on a leading role for the structural capacity of the compressed one, that generally is ignored.

With reference to the compressed rod elastic stability in the structure in-plane, it has been found as the hypothesis of an effective length equal to the half of its geometrical length is admissible, in fact is enough a stretched rod with the area equal to the 2.2% of the compressed rod to manage it to buckle in the second mode.

The stretched rod carries on an important role with reference to the out-of-plane compressed rod stability too.

Its action, that depends from $\Delta F = P_E/N_s$, is considerable already for ΔF values near to the unit; in fact for $\Delta F = 1$ the Euler buckling load of the brace compressed rod is $P_{cr} =$

$2.89P_E$.

If the stretched rod stress is $N_s = 2.35P_E$ then $P_{cr} = 4P_E$ and the compressed rod buckles in the second mode.

It is necessary to note that as the stretched rod stress N_s , useful to impose the buckling in the second mode, is linked to P_E , the more P_E is high the more N_s must be high. So it is necessary of course to give always a look to the strength check of the stretched rod in order that the maximum stress doesn't exceed the steel design stress f_d .

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